ME 420	Professor John M. Cimb	ala Lecture 13
Today, we will:		
-	pple problem from last lecture – Part (b) discussion of normal shocks	
Continuation of the example.	mple from last lecture:	
Example: Converging-	-Diverging nozzle	
Given: Air flows from a	a very large	
tank through a convergi		Outlet
nozzle. The test section		Wall
0. The outlet of the test		
x = 0.60 m, where it is e	· · _ ` ` · _ ` · _ ` · _ ` · _ ` · _ ` · _ ` · _ ` · _ ` · ` ·	Axis x
back pressure $P_b = 50.0$	kPa. In the 0 m	
tank,		0.6 m
• $P_{0,\text{inlet}} = 220 \text{ kPa}$ (a		
• $T_{0,\text{inlet}} = 300 \text{ K}$		

The cross-sectional area is known as a function of axial distance *x*. (see previous lecture)

(a) <u>To do</u>: For isentropic flow through the converging-diverging nozzle (no shocks), calculate and plot Mach number and nondimensional pressure $P/P_{0,\text{inlet}}$ as functions of *x*.

<u>Solution</u>: We assumed that the back pressure is low enough such that the flow in the C-D nozzle is isentropic everywhere (no shock). Numbers from last lecture:

We calculate $P_b/P_{0,\text{inlet}} = 50/220 = 0.2273$ – we *assume* this back pressure is low enough that the flow is supersonic through the entire diverging section of the nozzle, without any normal shocks in the nozzle. *Is this assumption true?*

For case E of our notes, a normal shock sits right at the exit plane of the CD nozzle. We calculated that downstream of this shock, $P_2 = 93.89$ kPa. So, as long as the actual P_b is less than this value of P_2 , we are assured that our assumption was indeed correct. Here, $P_b = 50.0$ kPa $< P_2 = 93.89$ kPa. *Thus, we have verified our assumption*.

(b) <u>To do</u>: Calculate the average air speed at the throat and at the exit plane. Also calculate the mass flow rate through this CD nozzle.

Solution: To be completed in class.

At throat, M=1	$M = \frac{V}{a} \rightarrow V = Ma$ so $M = 1$
a = JYRT	$M = \frac{V}{a} \rightarrow V = Ma$ $T = \frac{T}{T_0} T_0 = \left(1 + \frac{v-1}{2}M^2\right) T_0 - \frac{3vv}{k}$

$$T = 250.0 \text{ k} \text{ e that}$$

$$V = M_{G} \rightarrow V_{\text{that}} = a^{*}$$

$$V = \int Y R_{T}^{*} = \int (1.4) (2t_{10} \frac{m^{2}}{f_{1}^{*} k}) (250 \text{ k})$$

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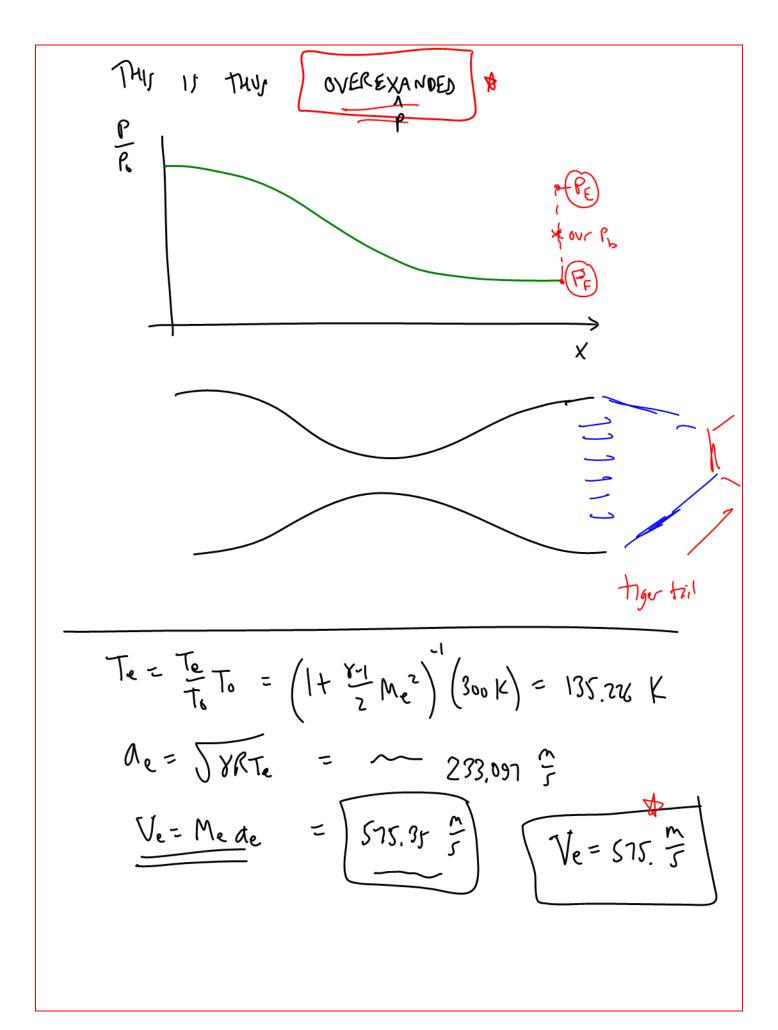
$$V = \int (1.4) (2t_{10} \frac{m^{2}}{f_{1}^{*} k}) (250 \text{ k})$$

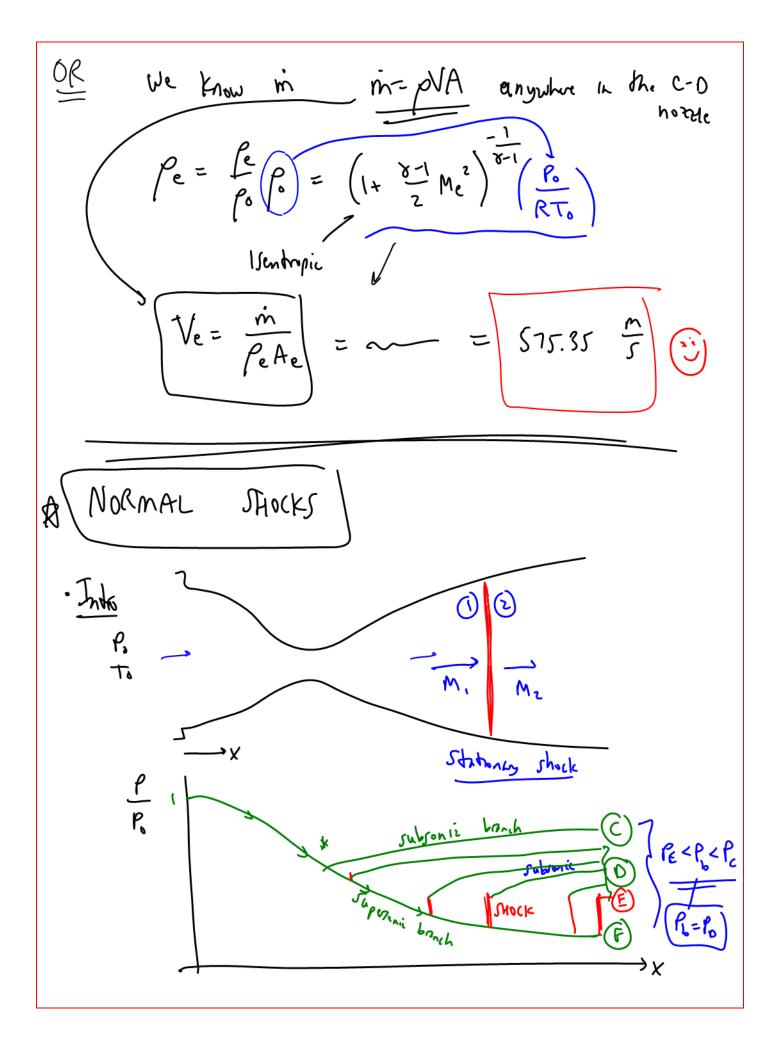
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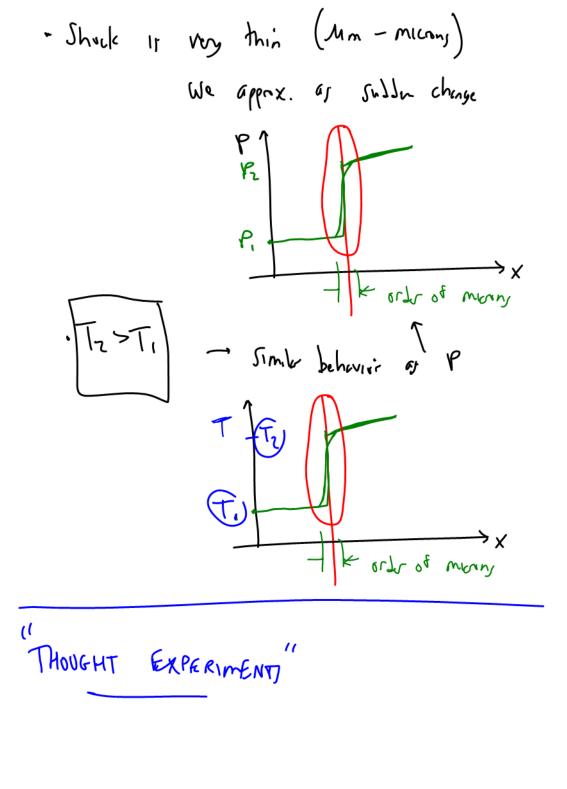
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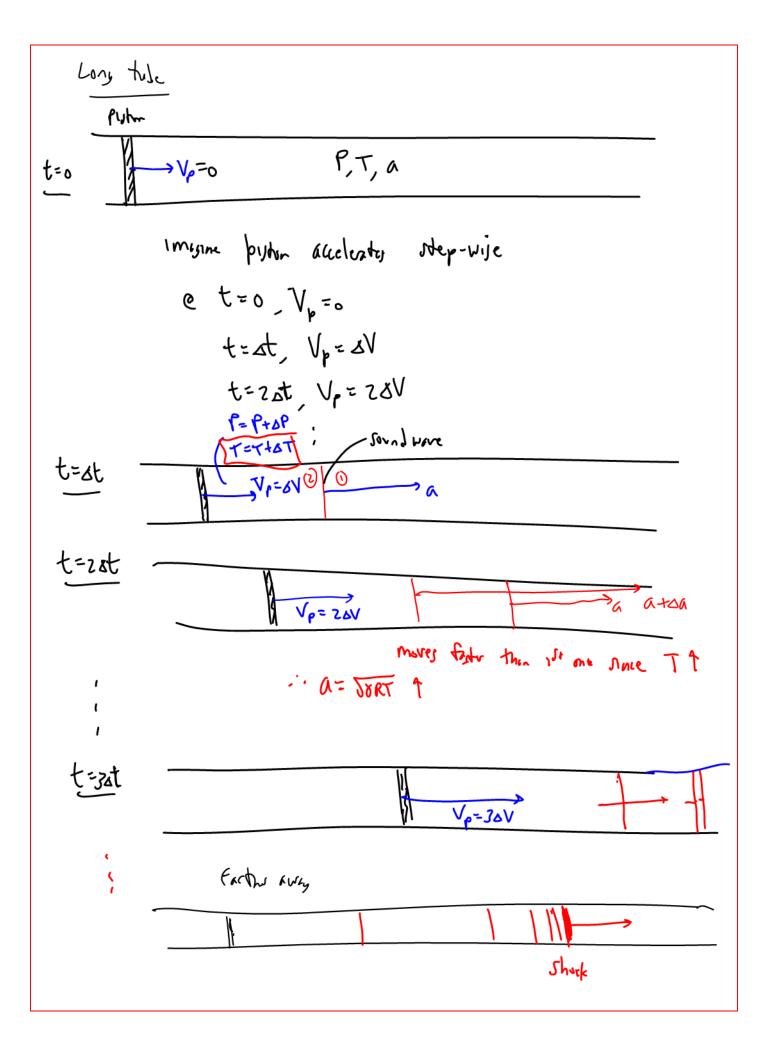
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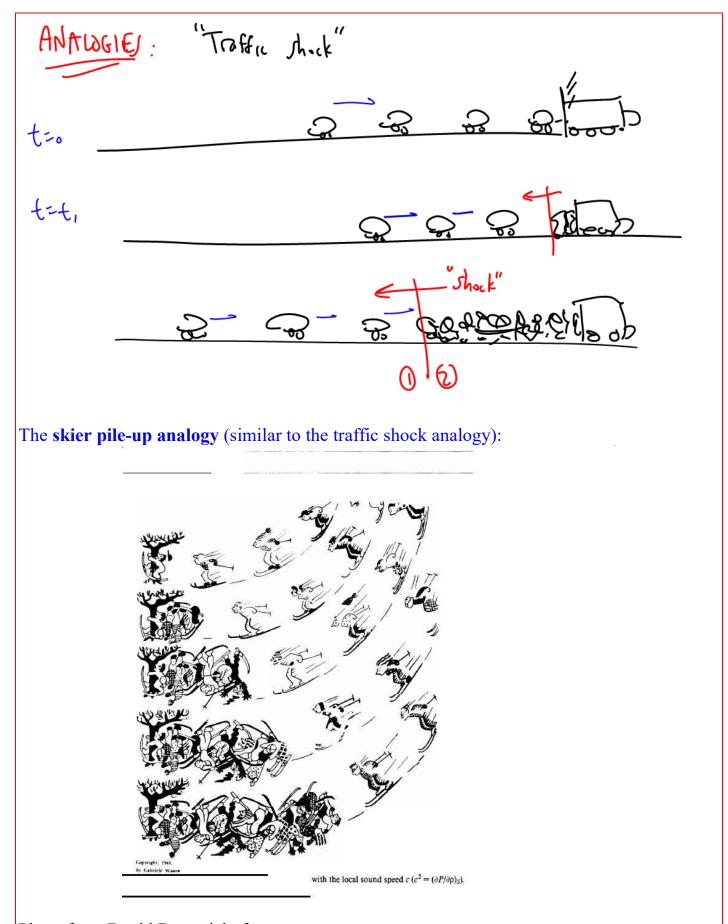
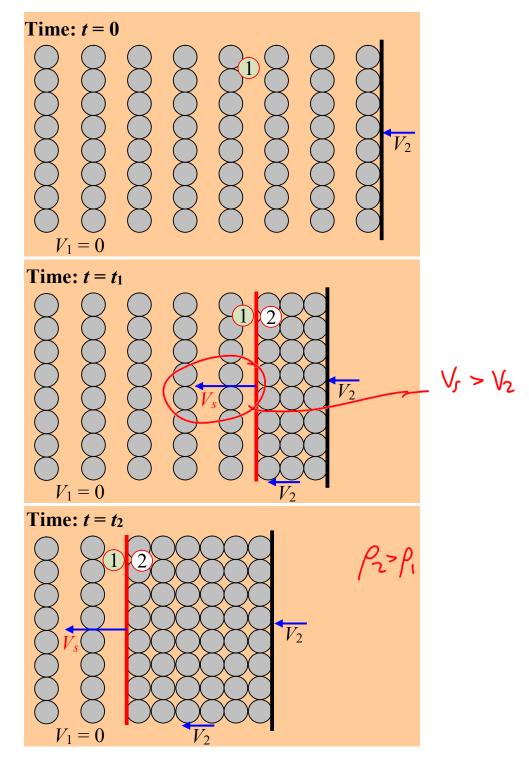


Photo from David Drewniak, from https://www.scribd.com/document/208788775/Shock-waves-vs-sounds-waves

The "*dime analogy*" (model a moving shock as rows of dimes that pile up when pushed by a rod or "piston" as sketched; three sequential times):



Comments:

- The vertical red line is analogous to a shock wave: $V_1 = 0$, $V_s > V_2$, $\rho_2 > \rho_1$ (there is sudden increase in density, and the "wave front" of dimes moves faster than the piston).
- The dimes in region 1 don't "know" anything is happening until the shock hits them.
- Similarly in a shock wave in air, the air in region 1 does not "know" anything is happening until the shock wave hits it.