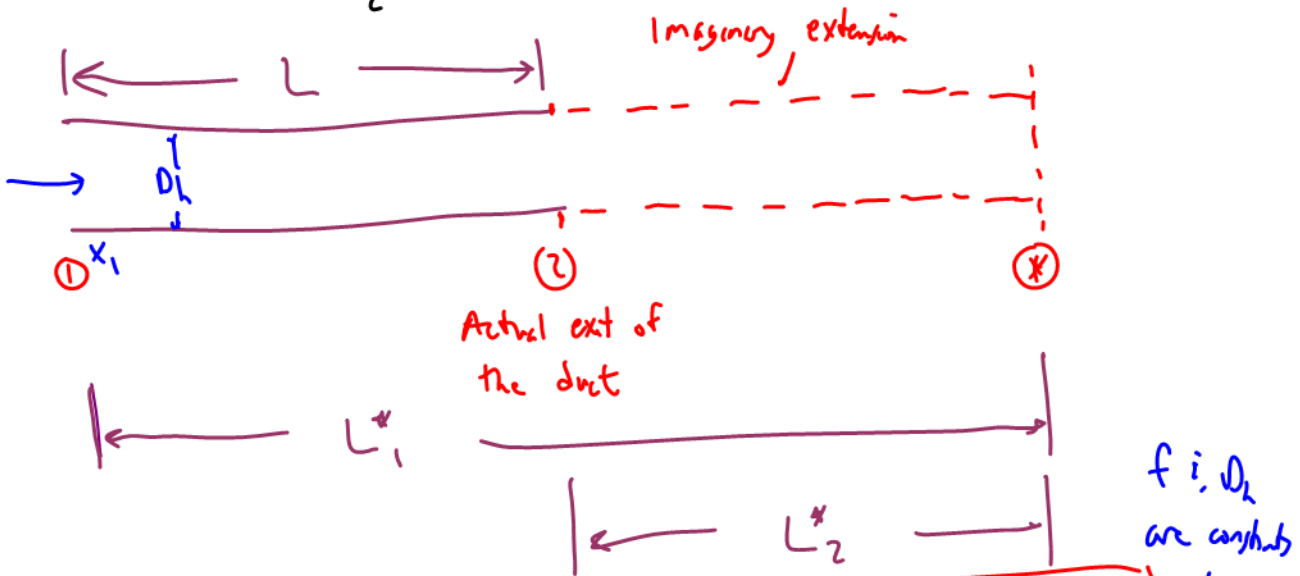


Define L_1^* : L_2^*



$$L = L_1^* - L_2^*$$

\Rightarrow

$$\frac{fL}{D_h} = \left(\frac{fL^*}{D_h} \right)_1 - \left(\frac{fL^*}{D_h} \right)_2$$

KEY

Eq (7) @ any location is

$$\frac{fL^*}{D_h} = \frac{1-M^2}{\gamma M^2} + \frac{\gamma+1}{2\gamma} \ln \left[\frac{(\gamma+1)M^2}{2+(\gamma-1)M^2} \right]$$

(8)

* workhorse eq. can apply @ (1), (2), anywhere

* Tabulated in the C.A.C. online

E.g. @ $M=0.3$; $\gamma=1.40 \rightarrow (8) \rightarrow \boxed{5.2993} = \frac{fL^*}{D_h}$
@ $M=0.3$

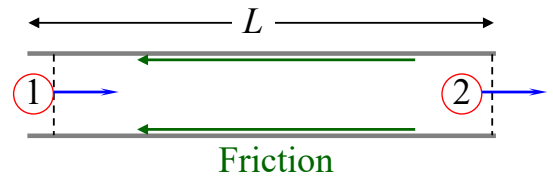
CAC's $\frac{4fL^*}{D} = \text{same as over } \frac{fL^*}{D_h}$

IGNORE THIS Y

E.g., $M = 2.0 \rightarrow \frac{fL^*}{D_h} = 0.305 \text{ @ } M = 2.0$

Example: Fanno flow (simple case)

Given: Air at $M_1 = 0.300$ enters a well-insulated (adiabatic) duct of length L and hydraulic diameter 0.250 m. The average Darcy friction factor through the length of the pipe is $0.0230 = f$



To do: Estimate the maximum possible duct length that will not affect the inlet conditions.

Solution:

Assumptions and Approximations (consistent with our simplified Fanno flow analysis):

1. The air is an ideal gas, and the properties do not change with temperature or pressure.
2. The flow is steady and one-D.
3. The flow is adiabatic but friction along the tube walls is *not* negligible.
4. The Darcy friction factor f is approximated as constant based on conditions at the inlet.

First we calculate

$$\left(\frac{fL_1^*}{D_h} \right) = \frac{1 - M_1^2}{\gamma M_1^2} + \frac{\gamma + 1}{2\gamma} \ln \left(\frac{(\gamma + 1)M_1^2}{2 + (\gamma - 1)M_1^2} \right) = 5.2993$$

To be completed in class.

$$L_{max} = L_1^* = \frac{5.2993 D_h}{f} = \frac{(5.2993)(0.250 \text{ m})}{0.0230}$$

$$L_{max} = 57.6 \text{ m}$$

★ PROCEDURE FOR GENERAL FANNO FLOW

Given



@ (1) Know $P_1, T_1, (M_1 \text{ or } V_1)$ → can calc ρ_1, μ_1, \dots (E known)

To do: Calc Flow properties @ (2)

(D_h known)

Soln: Assume $L < L^*$ (not choked)

PROCEDURE:

(1) Calc M_1 @ T_1 USE SUTHERLAND'S LAW (on eq sheet)

calc $Re_1 = \frac{\rho_1 V_1 D_h}{\mu_1} \quad \therefore \quad \frac{\varepsilon}{D_h}$

Use Churchill Eq. to get f_1

(2) Use Eq (8) @ ① to get $\left(\frac{fL^*}{D_h}\right)_1$

(3) Use our "key" Eq. to get

$$\left(\frac{fL^*}{D_h}\right)_2 = \left(\frac{fL^*}{D_h}\right)_1 - \left(\frac{fL}{D_h}\right)$$

(4) Use Eq (8) again to get M_2 INVERSELY

Implicit eq

choose subsonic answer

(5) Calc T_2 recall $T_0 = \text{const}$ (adiabatic)

$$T_2 = \frac{T_2}{T_0} \frac{T_0}{T_1} T_1 = \left(1 + \frac{\gamma-1}{2} M_2^2\right)^{-1} \left(1 + \frac{\gamma-1}{2} M_1^2\right) T_1$$

(6) Calc. other properties @ (2)

$$a_2 = \sqrt{\gamma R T_2}$$

$$V_2 = a_2 M_2$$

$$\rho_2 = \frac{\rho_1 V_1}{V_2}$$

$$P_2 = \rho_2 R T_2$$

$$\frac{P_{02}}{P_2} = \left(1 + \frac{\gamma-1}{2} M_2^2\right)^{\frac{\gamma}{\gamma-1}}$$

(M=1 @ *)

OR

$$\frac{T}{T^*} = \frac{T}{T_0} \frac{T_0}{T^*} = \left(1 + \frac{\gamma-1}{2} M^2\right)^{-1} \left(1 + \frac{\gamma-1}{2} (1)^2\right)$$

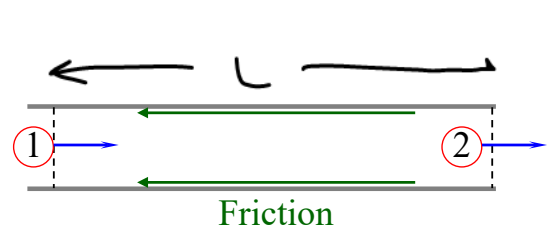
$$\frac{T}{T^*} = \frac{\gamma+1}{2 + (\gamma-1) M^2}$$

Similarly for $\frac{\rho_0^*}{\rho}$ $\frac{P}{P^*}$ etc.

Example: Fanno flow

Given:

- Air enters a 5.0-cm diameter, 27.0-m long tube at 450 K, 220 kPa, and 85.0 m/s. The average roughness height of the inside wall of the pipe is 0.08 mm. = ϵ
- The pipe is well-insulated, but we need to be concerned about friction in the pipe since it is so long.



To do: Estimate the temperature, pressure, velocity, and Mach number at location 2.

Solution:

Assumptions and Approximations (consistent with our simplified Fanno flow analysis):

- The air is an ideal gas, and the properties do not change with temperature or pressure.
- The flow is steady and one-D.
- The flow is adiabatic but friction along the tube walls is *not* negligible.
- The Darcy friction factor f is approximated as constant based on conditions at the inlet, and the Churchill equation is used to calculate f .

To be completed in class.

PROCEDURE:

(1) @ (1) $M_1 = \frac{V_1}{a_1} = \boxed{0.200 = M_1}$

Sutherland $\mu_1 = 2.499 \times 10^{-5} \frac{\text{kg}}{\text{m}\cdot\text{s}} @ T_1$

$Re_1 = \frac{\rho_1 V_1 D_h}{\mu_1} = 289,660$

With Re_1 & $\frac{\epsilon}{D_h} \rightarrow$ Can get $\boxed{f_1 = 0.02296}$
from Churchill Eq.

(2) $\left(\frac{fL^*}{D_h}\right)_1$ from Eq (8) $\rightarrow \boxed{\left(\frac{fL^*}{D_h}\right)_1 = 14.55}$

(3) "key" eq. $\left(\frac{fL^*}{D_h}\right)_2 = \left(\frac{fL^*}{D_h}\right)_1 - \left(\frac{fL}{D_h}\right)$
 $= 14.550 - 12.398 = 2.151$
 Actual $L = 27.0 \text{ m}$

(4) @ $\left(\frac{fL^3}{Dh}\right)_2 = 2.151 \rightarrow$ solve (8) inversedly
to get $M_2 = 0.4090$ *

(5) $T_2 = \frac{T_2}{T_0} \frac{T_0}{T_1} T_1 \rightarrow T_2 = 438.9 \text{ K}$ ✗

(6) $a_2 = \sqrt{\gamma R T_2} \rightarrow 171.8 \text{ m/s} = a_2$ This should be V2 not a2 (typo error)

$\rho_2 = \frac{p_1 V_1}{V_2} \rightarrow p_2 = \rho_2 R T_2 = 106.2 \text{ kPa}$

Alternate way:

$$\frac{T_1}{T^*} = \frac{\gamma + 1}{2 + (\gamma - 1) M_1^2} = 1.1905$$

$$\frac{T_2}{T^*} = M_2 = 1.1611$$

$$T_2 = \frac{T_2}{T^*} \frac{T^*}{T_1} T_1 = 438.9 \text{ K} \quad \text{😊}$$

OPTIONAL STEP (7) \rightarrow since $T_2 \neq T_1$, $M_2 \neq M_1$, $Re_2 \neq Re_1$

$$\therefore f_2 \neq f_1$$

we assumed $f = \text{constant}$ from ① to ②

Calc f_2 using Churchill @ $Re_2 \therefore \frac{\epsilon}{Dh}$

I get $\underline{f_2 = 0.02293}$

recall $\underline{f_1 = 0.02296}$

Calc avg. f $f = \frac{f_1 + f_2}{2}$

Repeat the analysis